

# 5

## **Chapter Five**

### **Mechanical Design**

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- 5.1 Design of Conveyor**
- 5.2 Calculating the Torque of the Conveyor**
- 5.3 Calculating the Power of the Motor**
- 5.4 Final Design Machine**

## 5.1 Design of Conveyor

### 5.1.1 Design Bearing of Conveyor

In the design of conveyor bearing we consider that there is no thrust loading, and the loading is only radial.

**Step 1:** At first, we will compute  $F_x$  and  $F_y$  by applying static equilibrium equations to the shaft supported by the bearing.

$$S_f = S_i + v_i t + \frac{1}{2} a t^2 \quad \text{..... Equation (5.1)}$$

Where  $S_f$  : The distance from the middle to the end of the conveyor.

$S_i$  : The initial distance, and equal zero.

$v_i$  : Initial velocity of the belt.

$a$  : Acceleration of the belt.

**For Checkweigher:**

$$0.075 = \frac{1}{2} a t^2$$

$$45 \rightarrow 60 \text{ sec}$$

$$1 \rightarrow t \text{ sec}$$

$$0.15 = a(1.33)^2$$

$$t \text{ sec} = \frac{60}{45} = 1.33 \text{ sec}$$

$$a = \frac{0.15}{1.7689} = 0.084 \text{ m}^2/\text{sec}$$

**For Infeed and Outfeed Conveyor:**

$$0.20 = \frac{1}{2} a t^2$$

$$1.33 \text{ sec} \rightarrow 0.15 \text{ m}$$

$$t \text{ sec} \rightarrow 0.4 \text{ m}$$

$$0.40 = a(3.54)^2$$

$$t \text{ sec} = \frac{0.4 * 1.33}{0.15} = 3.54 \text{ sec}$$

$$a = \frac{0.40}{12.57} = 0.031 \text{ m}^2/\text{sec}$$

Where

$$F_x = m_1 * a$$

$F_x$ : Force on the x – axis.

$$F_y = m_1 * g + \frac{1}{2} m_2 * g$$

$F_y$ : Force on the y – axis.

$a$  : acceleration of the belt.

$m_1$ : mass of the object.

$m_2$ : mass of the roller.

**For Checkweigher:**

$$F_x = 5 * 0.084 = 0.42 \text{ N}$$

$$F_y = 5 * 9.81 + \frac{1}{2} * 0.5 * 9.81 = 51.5 \text{ N.}$$

**For Infeed and Outfeed Conveyor:**

$$F_x = 5 * 0.031 = 0.1589 \text{ N}$$

$$F_y = 5 * 9.81 + \frac{1}{2} * 0.5 * 9.81 = 51.5 \text{ N.}$$

**Step 2:** Now we find the resultant radial load ( $F_r$ )

**For Checkweigher:**

$$F_r = \sqrt{F_x^2 + F_y^2} = \sqrt{(0.42)^2 + (51.5)^2}$$

$$F_r = 51.5 \text{ N.}$$

$$\theta = \tan^{-1} \frac{F_y}{F_x} = \tan^{-1} \frac{51.5}{0.42} = 89.53^\circ$$

**For Infeed and Outfeed Conveyor:**

$$F_r = \sqrt{F_x^2 + F_y^2} = \sqrt{(0.1589)^2 + (51.5)^2}$$

$$F_r = 51.5 \text{ N.}$$

$$\theta = \tan^{-1} \left( \frac{F_y}{F_x} \right) = \tan^{-1} \left( \frac{51.5}{0.1589} \right) = 89.82^\circ$$

Where  $F_r$ : Force resultant.

## Specifying FD

The design load can be defined by:

$$F_D = a_f * V * F_r \dots\dots\dots \text{Equation (5.2)}$$

Where:  $a_f$  is application factor (Table 5-1), used because loads are often variable (non-steady) and may increase during operation.

$V$  rotation factor, takes into account whether the inner or outer race rotates ring outer rotating 1.2 ring inner rotating 0.1  $V$  usually the inner race of the bearing rotate (i.e.,  $V=1$ ).[8]

$V = 1$  (rotating inner ring).

$a_f = 1$  (machinery with no impact).

$F_D = 1 * 1 * 51.5 = 51.5 \text{ N}$ . **Are the same for Checkweigher, Infeed and Outfeed Conveyor.**

**Step 3:** Assuming the desired life ( $L_D$ ) and Reliability ( $R_D$ ), we find  $X_D$

$$X_D = \frac{L_{Dh}}{L_{10}} \dots\dots\dots \text{Equation (5.3)}$$

$$L_{Dh} = L_{Dh} * N * 60$$

$$\ddot{\theta} = \frac{a}{r}$$

Where:  $\ddot{\theta}$ : angular acceleration of the pulley.

$r$ : radius of the pulley.

$X_D$ : Life ratio.

$L_D$ : Desired life (revolutions).

$L_{10}$ : Rating life (revolutions) = 1 million rev.

$L_{Dh} = 30000$  (general industrial machinery). (Table 5-2)

$N$ : speed motor.

$$0.15\text{m} \rightarrow 0.05\text{m}$$

$$x \text{ rev} \rightarrow 1 \text{ rev}$$

$$x \text{ rev} = \frac{0.15}{0.05} = 3$$

$$= 3 \text{ rev/m}$$

$$0.05\text{m} \rightarrow 0.01\text{m}$$

$$y \text{ rev} \rightarrow 1 \text{ rev}$$

$$y \text{ rev} = \frac{0.05}{0.01} = 5$$

$$x_{\text{rev}} * y_{\text{rev}} = 3 * 5$$

$$= 15 \text{ turn of motor per piece.}$$

**For Checkweigher:**

Length of the conveyor = 0.15 m.

Diameter of the roller = 0.05 m.

Diameter of the rod = 0.01 m.

$$\ddot{\theta} = \frac{0.084}{0.01} = 8.4 \text{ m/s}$$

$$N = 15 \frac{\text{rev}}{\text{m}} * \frac{1\text{m}}{1.33 \text{ sec}} * \frac{60 \text{ sec}}{1 \text{ min}} = 676.67 \text{ rpm}$$

$$L_{Dh} = 30000 * 676.67 * 60 = 1218.006 * 10^6$$

$$X_D = \frac{1218.006 * 10^6}{1 * 10^6} = 1218.006 \text{Hp}$$

**For Infeed and Outfeed Conveyor:**

Length of the conveyor = 0.40 m.

Diameter of the roller = 0.05 m.

Diameter of the rod = 0.01 m.

$$\ddot{\theta} = \frac{0.031}{0.01} = 3.1 \text{ m/s}$$

$$N = 40 \frac{\text{rev}}{\text{m}} * \frac{1\text{m}}{3.54 \text{ sec}} * \frac{60 \text{ sec}}{1 \text{ min}} = 685.7 \text{ rpm}$$

$$L_{Dh} = 30000 * 685.7 * 60 = 1234.28 * 10^6$$

$$X_D = \frac{1234.28 * 10^6}{1 * 10^6} = 1234.28 \text{h}$$

$$0.40\text{m} \rightarrow 0.05\text{m}$$

$$x \text{ rev} \rightarrow 1 \text{ rev}$$

$$x \text{ rev} = \frac{0.40}{0.05} = 8 \\ = 3 \text{ rev/m}$$

$$0.05\text{m} \rightarrow 0.01\text{m}$$

$$y \text{ rev} \rightarrow 1 \text{ rev}$$

$$y \text{ rev} = \frac{0.05}{0.01} = 5$$

$$x \text{ rev} * y \text{ rev} = 8 * 5 = 40 \text{ turn of} \\ \text{motor per piece}$$

**Step 4:** Calculate the required catalog rating:

$$C_{10} = X_D^{1/a} * F_D$$

Where  $C_{10}$ : is Catalog rating.  
 $a = 3$  (ball bearing)

$$a = \begin{cases} 3 & \text{for ball bearing} \\ 3.33 & \text{for roller bearing} \end{cases}$$

**For Checkweigher:**

$$C_{10} = (1218.006)^{1/3} * 51.5 = 549.99\text{N} \rightarrow 0.54999\text{ KN}$$

**For Infeed and Outfeed Conveyor:**

$$C_{10} = (1234.28)^{1/3} * 51.5 = 552.43\text{N} \rightarrow 0.55243\text{KN}$$

**Step 5:** Check the catalog and select a suitable bearing from (Table 5-3)

**For Checkweigher:**

$$C_{10} = 0.54999\text{ KN}, \text{ Bore} = 10\text{ mm}, \text{ OD} = 30\text{ mm}.$$

**For Infeed and Outfeed Conveyor:**

$$C_{10} = 0.55243\text{ KN}, \text{ Bore} = 10\text{ mm}, \text{ OD} = 30\text{ mm}.$$



**Figure 5.1:** Angular Contact Ball Bearing

## 5.2 Calculating the Torque of the Conveyor

$$T = J_e \ddot{\theta} + F_r \cdot r \quad \dots\dots\dots \text{Equation (5.4)}$$

Where  $J_e$ : is equivalent moment inertia.

T: Torque.

### 5.2.1 Calculate the total moment inertia of checkweigher conveyor

- $J_{Bearing} = \frac{\pi}{2}(r_{out} - r_{inner})^4$  ..... Equation (5.5)

Where  $r_{out}$ : is outradius.

$r_{inner}$ : is innerradius.

<b>For checkweigher:</b>  $J_{Bearing} = \frac{\pi}{2}(0.028 - 0.01)^4$ $= 1.6489 * 10^{-7} Kg/m^2$	<b>For Infeed and Outfeed Conveyor:</b>  $J_{Bearing} = \frac{\pi}{2}(0.028 - 0.01)^4$ $= 1.6489 * 10^{-7} Kg/m^2$
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- $J_{object} = V^2 * m$  ..... Equation (5.6)

Where V: is velocity of conveyor.

<b>For Checkweigher:</b>  45 object → 60 sec. X object → 60 sec.  $X = \frac{45 * 60}{60}$ $= 45 \text{ object/s}$ $v = \frac{0.15}{1.33} = 0.1127 \text{ m/s}$	<b>For Infeed and Outfeed Conveyor:</b>  45 object → 60 sec. X object → 60 sec.  $X = \frac{45 * 60}{60}$ $= 45 \text{ object/s}$ $v = \frac{0.40}{3.5} = 0.1142 \text{ m/s}$
$J_{object} = 0.1127^2 * 55$ $= 0.0635 Kg/m^2$	$J_{object} = 0.1142^2 * 5$ $= 0.0653 Kg/m^2$

- $J_{Belt} = \frac{V * m_3}{w}$  ..... Equation (5.7)

Where  $m_3$ : is mass of the Belt.  
 $w$ : angular velocity.

<b>For Checkweigher:</b>  $w = \frac{2\pi N}{60}$  $w = \frac{2\pi * 676.67}{60}$  $= 70.8 \text{ s}^{-1}$	<b>For Infeed and Outfeed Conveyor:</b>  $w = \frac{2\pi N}{60}$  $w = \frac{2\pi * 685.7}{60}$  $= 71.8 \text{ s}^{-1}$
$J_{Belt} = \frac{0.1127 * 0.4}{70.8}$ $= 6.367 * 10^{-4} \text{ Kg/m}^2$	$J_{Belt} = \frac{0.1142 * 0.4}{71.81}$ $= 6.36211 * 10^{-4} \text{ Kg/m}^2$

- $J_{roller} = \frac{1}{4} m_4 r^2 + \frac{1}{3} m_4 l^2$  ..... Equation (5.8)

Where  $m_4$ : is mass of the roller.  
 $l$ : length of the roller.

<b>For checkweigher:</b>  $J_{roller} = \frac{1}{4} * 0.5 * (0.025)^2 + \frac{1}{3} * 0.5 * 0.14^2$  $= 3.344 * 10^{-3} \text{ Kg/m}^2$	<b>For Infeed and Outfeed Conveyor:</b>  $J_{roller} = \frac{1}{4} * 0.5 * (0.025)^2 + \frac{1}{3} * 0.5 * 0.14^2$  $= 3.344 * 10^{-3} \text{ Kg/m}^2$
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**For Checkweigher:**

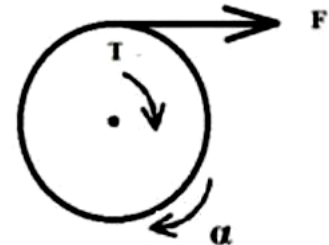
$$J_e = 2 * J_{roller} + 4 * J_{Bearing} + J_{object} + J_{Belt} \dots\dots\dots \text{Equation (5.9)}$$

$$J_e = (2 * 3.344 * 10^{-3}) + (4 * 1.6489 * 10^{-7}) + (0.0635) + (6.367 * 10^{-4})$$

$$= 0.0708 \text{ Kg/m}^2$$

$$T = J_e \ddot{\theta} + F_r \cdot r \dots\dots\dots \text{Equation (5.10)}$$

$$= 0.0708 * 8.4 + 51.5 * 0.01 = 1.109 \text{ N.m}$$



**Figure 5.2:** Free body  
Diagram of Cylinder

**For Infeed and Outfeed Conveyor:**

$$J_e = J_{pulley} + 2 * J_{roller} + 4 * J_{Bearing} + J_{object} + J_{Belt}$$

$$J_e = (1.5707 * 10^{-8}) + (2 * 3.344 * 10^{-3}) + (4 * 1.6489 * 10^{-7})$$

$$+ (0.0653) + (6.36211 * 10^{-4}).$$

$$= 0.07262 \text{ Kg/m}^2$$

$$J_{e(total)} = J_e * 2$$

$$J_{e(total)} = 0.07262 * 2 = 0.1452 \text{ Kg/m}^2$$

$$T = J_{e(total)} \ddot{\theta} + F_r \cdot r$$

$$= 0.1452 * 3.1 + 51.5 * 0.01 = 0.9652 \text{ N.m}$$

### 5.3 Calculating the Power of the Motor

$$P_{out} = w.T \dots\dots\dots \text{Equation (5.11)}$$

$$H_p = \frac{P_{out}}{746} \dots\dots\dots \text{Equation (5.12)}$$

$$H_{p(safty\ factor)} = \alpha * H_p$$

Where  $H_p$ : is hourse power.

$\alpha$ : isasaftyfactor equal 1.13

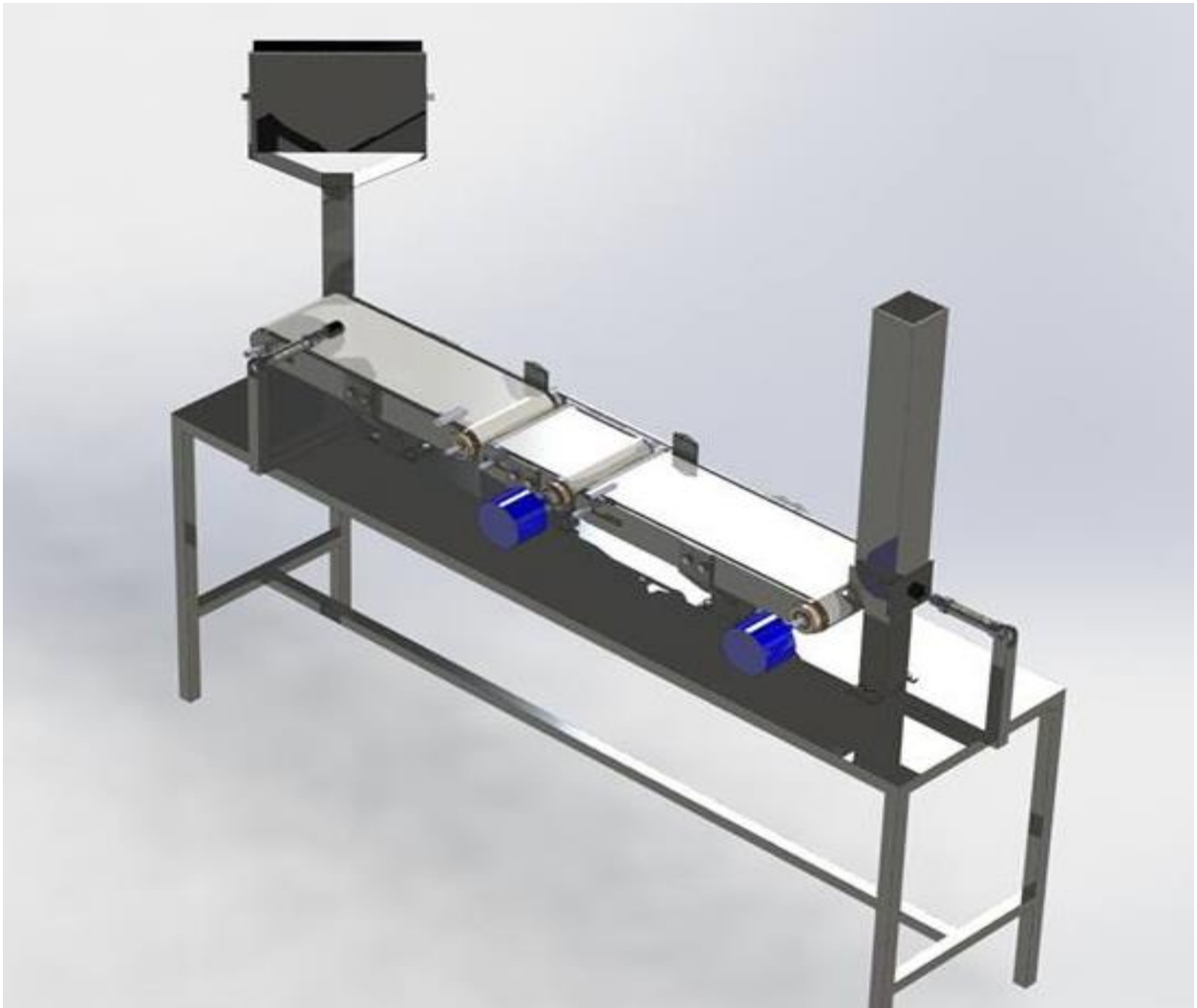
For Checkweigher:	For Infeed and Outfeed Conveyor:
$p_{out} = 70.8 * 1.109 = 78.58 \text{ Watt.}$	$p_{out} = 71.81 * 0.9652 = 69.31 \text{ Watt.}$
$H_p = \frac{78.58}{746} = 0.105 H_p$	$H_p = \frac{69.31}{746} = 0.0929 H_p$
$H_{p(safty\ factor)} = 1.13 * 0.105$ $= 0.119 H_p$	$H_{p(safty\ factor)} = 1.13 * 0.0929$ $= 0.104 H_p$

After we make a calculation, we choose a motor 0.18 HP, 1500 rpm, 3-phase as shown in figure



**Figure 5.3:** Electrical Motor

## 5.4 Final Design Machine



**Figure 5.4:** Final Design Machine